

Acoustic Levitation from Superposition of Spherical Harmonics Expansions of Elementary Sources: Analysis of Dependency on Wavenumber and Order

Carl Andersson

Division of Applied Acoustics
Chalmers University of Technology
Gothenburg, Sweden
Email: carl.andersson@chalmers.se

Jens Ahrens

Division of Applied Acoustics
Chalmers University of Technology
Gothenburg, Sweden
Email: jens.ahrens@chalmers.se

Abstract—Acoustic levitation of spherical objects in mid-air is a technique that is gaining traction in various human computer interaction applications such as data visualization or interactive displays. The commonly used hardware platforms are phased ultrasonic transducer arrays, used to levitate small spheres of polystyrene. Previous works have used angular spectrum decomposition of the incident sound field to derive the radiation force exerted on an arbitrary sphere. We show an alternate formulation more suited to sound fields from transducer arrays, based on direct spherical harmonics expansions of the sound fields from the individual transducer elements in the array. Finally we investigate how the truncation order of the spherical harmonics expansion influence the calculated force, for varying sphere sizes.

I. INTRODUCTION

Dexterous manipulation of the levitating objects in real-time requires fast calculation of the phases of the elements in the array. This is typically achieved using either simple, limited models of the radiation force along with numerical optimization, or empirical direct expressions for the element phases [1]. These methods require the levitating object to be small enough that Rayleigh scattering can be applied, i.e., $ka < 1$ where a is the radius of the sphere and $k = \omega/c$ is the wavenumber of the impinging waves [2]. Levitation of larger objects has previously been done using computationally expensive models to calculate the radiation force, which prevents practical real-time manipulation of objects [3].

We propose spherical harmonics based modeling of the radiation force, where the expansion coefficients are calculated directly from the known radiating elements in the array. Accurate radiation force calculation requires expansion of the pressure field from each element at, in principle, infinite orders of spherical harmonics. This is naturally not possible, and for a spherical object of finite radius there is a limiting value to which the expansions at lower orders will converge to. Using our formulation, we investigate the relation between expansion order, sphere radius, and resulting force. This is done using a

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simulation of a 16x16-element array of ultrasonic transducers, which is similar to the arrays used in previous works [4, 5, 6]. The elements are phased with a static scenario known, based on simpler models, to produce stable acoustic levitation of small beads at arbitrary positions above the array.

In this paper we describe our sound field model and subsequent radiation force model in section II, followed by results from numerical simulations to investigate the relation between bead size and required spherical harmonics expansion order in section III.

II. METHOD

Our method is composed of two parts: how to efficiently obtain the spherical harmonics expansion of a sound field generated by a phased array using elementary sources, and how to calculate the radiation force on a spherical object in an arbitrary sound field using spherical harmonics expansion. Note that the last part is not dependent on the first, and can be applied to any sound field of which the spherical harmonics expansion is known.

A. Sound field decomposition

The sound field p_i impinging on a body can be written as a sum of spherical harmonics and spherical Bessel functions, as [7]

$$p_i(\vec{r}) = \sum_{n=0}^{\infty} \sum_{m=-n}^n j_n(kr) S_{n,m} Y_{nm}(\theta, \varphi) \quad (1)$$

where \vec{r} is the position vector in the field, $S_{n,m}$ are the expansion coefficients, and the time dependency $e^{-i\omega t}$ is implicit. The spherical harmonics bases Y_{nm} are given as

$$Y_{nm}(\theta, \varphi) = \sqrt{\frac{2n+1}{4\pi} \frac{(n-m)!}{(n+m)!}} P_n^m(\cos \theta) e^{im\varphi}, \quad (2)$$

where the associated Legendre polynomials P_n^m are given as

$$P_n^m(x) = (-1)^m (1-x^2)^{m/2} \frac{d^m P_n(x)}{dx^m}, \quad (3)$$

and all coordinates are taken relative to the center of the levitating bead. In a similar way the scattered sound field can be written as

$$p_s(\vec{r}, \omega) = \sum_{n=0}^{\infty} \sum_{m=-n}^n h_n(kr) \hat{S}_{n,m} Y_{nm}(\theta, \varphi), \quad (4)$$

where $h_n(kr) = h_n^{(1)}(ka) = j_n(kr) + i y_n(kr)$ are the spherical Hankel functions of the first kind. If the scattering body is a solid sphere of radius a the coefficients $\hat{S}_{n,m}$ of the scattered sound field can be found as [8]

$$\hat{S}_{n,m} = - \frac{j_n(ka) j'_n(k_* a) - \tilde{Z} j'_n(ka) j_n(k_* a)}{h_n(ka) j'_n(k_* a) - \tilde{Z} h'_n(ka) j_n(k_* a)} S_{n,m} = c_n S_{n,m} \quad (5)$$

where \tilde{Z} is the relative impedance $(\rho_* c_*)/(\rho_0 c_0)$, subscript $*$ indicating the material of the sphere and subscript 0 indicate the medium, and primed symbols indicate derivatives with respect to the argument.

For an arbitrary incident sound field the expansion coefficients $S_{n,m}$ can be calculated using a spherical integration surface Ω , as

$$S_{n,m} = \frac{1}{j_n(kr_0)} \int_{\Omega} p_i(\vec{r}) Y_{nm}^*(\theta, \varphi) d\Omega \quad (6)$$

where r_0 is the radius of the spherical integration surface. Note that if this equation is applied directly it is essential that the radius of the integration surface is chosen to avoid zeros of the spherical Bessel functions $j_n(kr_0)$. If the sound pressure field is created by a discrete transducer array it can be written as the superposition of the contributions p^j from each element j . Since both the summation and the integral for the coefficients $S_{n,m}$ converge uniformly we can exchange the order of summation and integration as

$$S_{n,m} = \sum_j \frac{1}{j_n(kr_0)} \int_{\Omega} p^j(\vec{r}) Y_{nm}^*(\theta, \varphi) d\Omega = \sum_j S_{n,m}^j \quad (7)$$

i.e. the expansion coefficients of the superposed incident sound field are given by the summation of the expansion coefficients for each element in the transducer array.

Each transducer element is modeled as a point source, generating the pressure

$$p^j(\vec{r}) = \frac{q^j}{|\vec{r} - \vec{r}^j|} e^{ik|\vec{r} - \vec{r}^j|} \quad (8)$$

where \vec{r}^j is the location of element j and q^j is the complex source strength for the element. Modeling the transducers as point sources is a convenient choice due to the simple expressions for the expansions coefficients, see below, but there are ways to include the directivity of the transducer as well [9]. The expansion coefficients for the above expression are known to be [7]

$$S_{n,m}^j = q^j 4\pi i k h_n(kr^j) Y_{nm}^*(\theta^j, \varphi^j) \quad (9)$$

in the region $r < r^j$, i.e. closer to the scatterer than the transducer element. Note that this formulation does not require

choosing a suitable radius for an integration surface, in contrast to the direct application of (6). Equations (7) and (9) are the main equations for our sound field model, and the reason why our approach is suited for transducer arrays.

B. Radiation force from spherical harmonics expansions

We use a method very similar to Sapozhnikov and Bailey to calculate the radiation force, with differences in the description of the sound field [10]. Their approach is to use a plane wave decomposition of the sound field, solving the scattering problem using spherical harmonics decompositions of each plane wave separately. As shown above, we use a solution of the scattering problem where a spherical harmonics decomposition is used directly, without any plane wave decomposition. We also use a simpler model for the scattering coefficients c_n for our computations, but it is very easy to use other formulations of the scattering coefficients, e.g. [11, 12]. Modifying the derivations in section II D in the above mentioned paper with our description of the sound field gives the following result for the radiation force

$$\begin{aligned} F_x &= \frac{1}{8\rho_0 c_0^2 k^2} \Re \left\{ \sum_{n=0}^{\infty} \sum_{m=-n}^n \Psi_n A_{nm} \cdot (S_{n,m} S_{n+1,m+1}^* - S_{n,-m} S_{n+1,-m-1}^*) \right\} \\ F_y &= \frac{1}{8\rho_0 c_0^2 k^2} \Im \left\{ \sum_{n=0}^{\infty} \sum_{m=-n}^n \Psi_n A_{nm} \cdot (S_{n,m} S_{n+1,m+1}^* + S_{n,-m} S_{n+1,-m-1}^*) \right\} \\ F_z &= \frac{1}{8\rho_0 c_0^2 k^2} \Re \left\{ \sum_{n=0}^{\infty} \sum_{m=-n}^n \Psi_n B_{nm} (S_{n,m} S_{n+1,m}^*) \right\} \end{aligned} \quad (10)$$

where

$$\begin{aligned} \Psi_n &= i(1 + 2c_n)(1 + 2c_{n+1}^*) - i \\ &= 2i(c_n + c_{n+1}^* + 2c_n c_{n+1}^*) \\ A_{nm} &= \sqrt{\frac{(n+m+1)(n+m+2)}{(2n+1)(2n+3)}} \\ B_{nm} &= -2\sqrt{\frac{(n+m+1)(n-m+1)}{(2n+1)(2n+3)}} \end{aligned} \quad (11)$$

which differs from the original result only in the definition of the expansion coefficients $S_{n,m}$ and the coefficients Ψ_n . Equations (10) and (11), supported by (7) and (9) are the main theoretical results of this paper.

C. Comparison with previous works

This section is a short comparison with the original equations derived by Sapozhnikov and Bailey [10]. To recover the

original expressions, take the incident pressure as a superposition of plane waves

$$p_i(\vec{r}) = \frac{1}{4\pi^2} \iint_{k_x^2 + k_y^2 \leq k^2} S(k_x, k_y) e^{i(k_x x + k_y y + \sqrt{k^2 - k_x^2 - k_y^2} z)} dk_x dk_y \quad (12)$$

which can be expanded as in (1) by using the addition theorem for spherical harmonics to obtain the expansion coefficients [7]

$$S_{n,m} = \frac{i^n}{\pi} \iint_{k_x^2 + k_y^2 \leq k^2} S(k_x, k_y) Y_{nm}^*(\theta_k, \varphi_k) dk_x dk_y \quad (13)$$

where θ_k, φ_k denote the incidence angle corresponding to k_x, k_y , compare with (34) in [10]. Inserting this expression in (10) directly reduces to the expressions in the original work.

III. RESULTS

A flat transducer array with 16×16 elements in a square grid with a 1 cm spacing between the element centers is used for all simulations. Each transducer element is modeled as a point source, see (8), operating at 40 kHz and with a maximum amplitude of $|q^j| \leq 6 \text{ N/m}$. The array phases are chosen to produce a sound field capable of capturing a small bead, by focusing all transducers at the desired location but including an additional phase shift of π radians for one side of the array [5].

The radiation force is calculated using (10) for varying radii a corresponding to ka between 0.05 to 50 up to order $n = 50$. The target levitation location is 80 mm above the center of the array, and the forces are calculated 1 mm offset in each axis. The calculations are not performed in the center of the trap since the symmetry of the sound field causes the horizontal forces to be zero in the center of the trap.

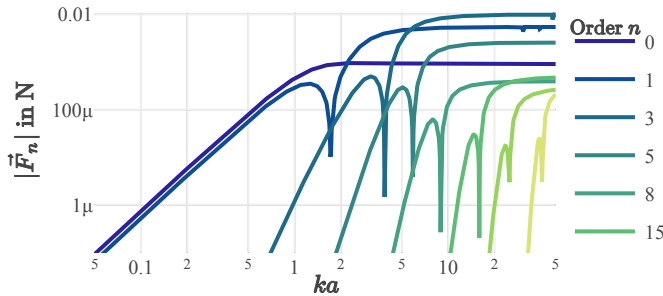


Fig. 1. Magnitude of force components from representative orders, shown over ka by varying the radius of the target sphere.

From (10) we identify the partial force components \vec{F}_{nm} and the force component \vec{F}_n of a given order n , as

$$\vec{F} = \sum_{n=0}^{\infty} \sum_{m=-n}^n \vec{F}_{nm} = \sum_{n=0}^{\infty} \vec{F}_n. \quad (14)$$

The magnitude of the force components $|\vec{F}_n|$ are shown for a few representative orders in Fig. 1. In a practical implementation the infinite sum needs to end after a finite number of

orders $n \leq N$. The total force vector at a given truncation order N , i.e.

$$\vec{F}_N = \sum_{n=0}^N \vec{F}_n \quad (15)$$

is shown in Fig. 2 for a few orders.

From Fig. 1 it is clear that the higher order components do not play a role for smaller spheres. For $ka < n$ the force magnitude is increasing with $(ka)^{2n+1}$, with the exception of $n = 0$ which behaves like $n = 1$. This behavior is expected from a small argument approximation the scattering coefficients c_n and the coefficients Ψ_n . Similar behavior can be seen from the varying truncation orders of the force vector in Fig. 2. Comparing the lower order truncations with the higher order truncations show that a truncation of order N is consistent with the higher order truncations up to $ka \approx N$, similar to results obtained in [11, pp. 427]. We can also see many resonances in the large ka domain, and that for the simulated case the higher order components act to smooth out the lower order components to some extent.

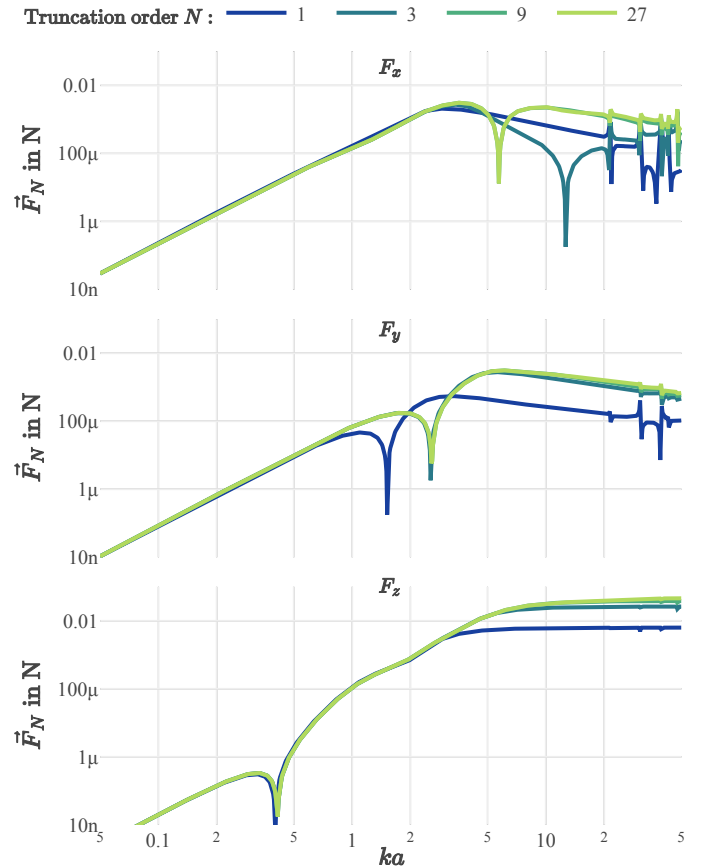


Fig. 2. Total force vector for different truncation orders N .

IV. CONCLUSIONS

We have shown a fast explicit method for calculating the radiation force on spherical objects in a sound field generated

by e.g. phased arrays. Our simulations indicate that for a given radius of the spherical object it is sufficient to truncate the spherical harmonics domain summation at approximately order $ka = N$, which confirms expectations based on previous works.

The approach is easily optimized for repeated calculation of the radiation force at a given location for different settings of the phased array by storing the constant values $S_{n,m}^j/q_j$ for each element in the array, as well as the coefficients in (11). The radiation force for a set of array element amplitudes and phases \vec{q} is then calculated using simple additions and multiplications of the pre-calculated values, and no Bessel functions or spherical harmonics are evaluated. This is a structure highly suited for numerical optimizations using the array settings as the parameters of a cost function, and can be used to numerically determine array configurations to fulfill some condition, similar to what is done in [4].

REFERENCES

- [1] Themis Omirou et al. “LeviPath: Modular Acoustic Levitation for 3D Path Visualisations”. In: *Proceedings of the 33rd Annual ACM Conference on Human Factors in Computing Systems - CHI '15* (Apr. 2015), pp. 309–312.
- [2] Henrik Bruus. “Acoustofluidics 7: The acoustic radiation force on small particles”. In: *Lab on a Chip* 12.6 (2012), pp. 1014–1021.
- [3] Seki Inoue et al. “Acoustical boundary hologram for macroscopic rigid-body levitation”. In: *Journal of the Acoustical Society of America* 145.1 (Jan. 2019), pp. 328–337.
- [4] Carl Andersson and Jens Ahrens. “A Method for Simultaneous Creation of an Acoustic Trap and a Quiet Zone”. In: *2018 IEEE 10th Sensor Array and Multichannel Signal Processing Workshop (SAM)*. IEEE, 2018, pp. 622–626.
- [5] Asier Marzo et al. “Holographic acoustic elements for manipulation of levitated objects”. In: *Nature Communications* 6.1 (2015), p. 4316.
- [6] Carl Andersson and Jens Ahrens. “Minimum Trap Separation for Acoustical Levitation Using Phased Ultrasonic Transducer Arrays”. In: *Proceedings of the 23rd International Congress on Acoustics*. Aachen, Germany, Sept. 9–13, 2019, pp. 1117–1123.
- [7] Earl G Williams. *Fourier Acoustics*. Sound Radiation and Nearfield Acoustical Holography. Elsevier, June 1999.
- [8] M F Hamilton and D T Blackstock. *Nonlinear acoustics*. Acoustical Society of America, 2008.
- [9] Jens Ahrens. *Analytic Methods of Sound Field Synthesis*. Berlin, Heidelberg: Springer Berlin Heidelberg, Jan. 1, 2012. ISBN: 978-3-642-25742-1.
- [10] Oleg A Sapozhnikov and Michael R Bailey. “Radiation force of an arbitrary acoustic beam on an elastic sphere in a fluid”. In: *Journal of the Acoustical Society of America* 133.2 (Feb. 2013), pp. 661–676.
- [11] N A Gumerov and R Duraiswami. *Fast Multipole Methods for the Helmholtz Equation in Three Dimensions*. Elsevier, Jan. 1, 2004. ISBN: 0-08-044371-0.
- [12] James J Faran Jr. “Sound Scattering by Solid Cylinders and Spheres”. In: *Journal of the Acoustical Society of America* 23.4 (June 18, 2005), pp. 405–418. DOI: 10.1121/1.1906780.